ARMY RESEARCH LABORATORY

A Two-Dimensional Meteorological Computer Model for the Forest Canopy

by Arnold Tunick

ARL-MR-569 **August 2003**

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Arnold Tunick
Computational and Information Sciences Directorate, ARL

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14. ABSTRACT

This report presents the equation set, modeling assumptions, and some initial results from a new, physics-based computer model that is being developed for two-dimensional forest canopy wind flow, temperature, and turbulence calculations. The model is based on the conservation (simplified Navier-Stokes) equations for continuity, momentum, Reynolds stress, energy, heat flux, and turbulent temperature variance. A set of simultaneous equations for each of 12 computed variables is solved iteratively on a computational grid consisting of 10 × 60 points. Horizontal grid spacing is 50 m, and vertical grid spacing is 0.5 m. The model domain is 500 × 30 m. It is anticipated that improved turbulence and micrometeorological models for forest canopies will become increasingly useful for military acoustic application research.

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1. Introduction

In a previous study (*1, 2*), it was found that a useful mathematical representation of the wind flow, temperatures, and turbulence inside and above a (uniform) continuous forest stand could be obtained by means of a one-dimensional (1-D), steady-state, second-order turbulence closure model, with an embedded radiative transfer and energy budget algorithm to predict the heat source. Development of this model made it possible to generate realistic profiles for effective sound speed inside and above a forest canopy. In turn, these data were used as input to an acoustic propagation model that predicts atmospheric and terrain effects on short-range acoustic attenuation (*3, 4*). As a result, it was shown that attenuation and "ducting" of acoustic waves in and around forests is significantly influenced by local micrometeorological profile structure.

However, forest stands are typically inhomogeneous, containing nonuniform distributions of canopy height and leaf area density (*5*). In addition, open fields, roadways, and buildings often border forests. Hence, to begin to address nonuniform forests and forest edges, this report presents the equation set, modeling assumptions, and some initial results from a new, physics-based computer model that is being developed for two-dimensional (2-D) forest canopy wind flow, temperature, and turbulence calculations. Like the earlier 1-D model, the 2-D model is based on the conservation (simplified Navier-Stokes) equations for continuity, momentum, Reynolds stress, energy, heat flux, and turbulent temperature variance. However, in this case, a set of simultaneous equations for each of 12 computed variables is solved iteratively on a computational grid consisting of 10×60 points. Horizontal grid spacing is 50 m and vertical grid spacing is 0.5 m. The model domain is 500×30 m. It is anticipated that improved physics-based theory and computer modeling for meteorology coupled to acoustics will become increasingly useful to predict effective sound speed information for military acoustic application research (*6, 7*).

The mathematical model for the forest canopy is described in section 2. Initial model results for uniform forest stands are shown in section 3.1. Initial model results for nonuniform forest stands, i.e., those that contain a single step change in canopy height, are shown in section 3.2. A summary and conclusions are provided in section 4.

2. Forest Canopy Model

2.1 Conservation Equations

The conservation (simplified Navier Stokes) equations for the current model, neglecting coriolis* forces, can be expressed as follows:

*Other than a few authors *(8–10)*, most consider the effect of the coriolis force as being negligible for the scales of motion considered.

The continuity equation is

$$-\frac{1}{\langle\overline{\rho}\rangle}\frac{\partial\langle\overline{\rho}\rangle}{\partial t}=0=\frac{\partial\langle\overline{u_i}\rangle}{\partial x_i}. \tag{1}$$

The momentum equation is

$$\frac{\partial\langle\overline{u_i}\rangle}{\partial t}=0=-\left\langle\overline{u_j}\frac{\partial\overline{u_i}}{\partial x_j}\right\rangle-\frac{\partial\left\langle\overline{u_i'u_j'}\right\rangle}{\partial x_j}-\left(\left\langle\frac{\partial\overline{p}}{\partial x_i}\right\rangle+\left\langle\frac{\partial\overline{p''}}{\partial x_i}\right\rangle\right)+\frac{g\theta_0}{T}\delta_{i3}+\nu\left(\frac{\partial^2\langle\overline{u_i}\rangle}{\partial x_j\partial x_j}+\left\langle\frac{\partial^2\overline{u''}}{\partial x_j\partial x_j}\right\rangle\right). \tag{2}$$

Here, t is the independent variable time, $\overline{\rho}$ is air density, $\overline{u_i}$ is the *i-component* of the wind velocity vector ($u_1=\overline{u},\ u_2=\overline{v},\ u_3=\overline{w}$), and x_i is the *i-component* of the position vector ($x_1=x$, $x_2=y$, $x_3=z$). In addition, p is static pressure normalized by air density (i.e., kinematic pressure), g is the acceleration due to gravity, T is the absolute temperature, θ_0 is a deviation from a reference temperature that decreases with height at the adiabatic lapse rate, and ν is kinematic viscosity. The overbar and primed variables indicate the mean (time averaged) and fluctuating components of the given quantity, whereas the brackets, $\langle\ \rangle$, and double primed variables indicate horizontal averaging and departures from the horizontal averaging operator *(11)*.

The stress equation for non-adiabatic conditions is

$$\begin{aligned}\frac{\partial\left\langle\overline{u_i'u_k'}\right\rangle}{\partial t}=0=&-\langle\overline{u_j}\rangle\frac{\partial\left\langle\overline{u_i'u_k'}\right\rangle}{\partial x_j}-\left(\left\langle\overline{u_j'u_k'}\right\rangle\frac{\partial\langle\overline{u_i}\rangle}{\partial x_j}+\left\langle\overline{u_i'u_j'}\right\rangle\frac{\partial\langle\overline{u_k}\rangle}{\partial x_j}\right)-\left(\frac{\partial\left\langle\overline{u_i'u_j'u_k'}\right\rangle}{\partial x_j}\right)+\frac{g}{T}\left(\left\langle\overline{u_i'\theta'}\right\rangle\delta_{k3}+\left\langle\overline{u_k'\theta'}\right\rangle\delta_{i3}\right)\\&-\left(\left\langle\overline{u_k'\frac{\partial p'}{\partial x_i}}\right\rangle+\left\langle\overline{u_i'\frac{\partial p'}{\partial x_k}}\right\rangle\right)+\left(\langle\overline{u_k}\rangle\left\langle\frac{\partial\overline{p''}}{\partial x_i}\right\rangle+\langle\overline{u_i}\rangle\left\langle\frac{\partial\overline{p''}}{\partial x_k}\right\rangle\right)+\upsilon\left(-2\left\langle\overline{\frac{\partial u_i'}{\partial x_j}\frac{\partial u_k'}{\partial x_j}}\right\rangle+\frac{\partial^2\left\langle\overline{u_i''u_k''}\right\rangle}{\partial x_j\partial x_j}\right).\end{aligned} \tag{3}$$

The energy equation is

$$\frac{d\langle\overline{\theta}\rangle}{dt}=0=-\langle\overline{u_k}\rangle\frac{\partial\langle\overline{\theta}\rangle}{\partial x_k}-\frac{\partial\left\langle\overline{u_k'\theta'}\right\rangle}{\partial x_k}+\kappa_T\frac{\partial^2\langle\overline{\theta}\rangle}{\partial x_k\partial x_k}+S_\theta\quad. \tag{4}$$

Here, θ is ambient air temperature, κ_T is thermal diffusivity, and S_θ is the forest canopy heat source (or sink). The heat source can be expressed as a function of leaf surface-to-ambient-air temperature differences (*12*).

The heat flux equation is

$$\frac{\partial\left\langle\overline{u_j^{'}\theta^{'}}\right\rangle}{\partial t} = 0 = -\left\langle\overline{u_k}\right\rangle\frac{\partial\left\langle\overline{u_j^{'}\theta^{'}}\right\rangle}{\partial x_k} - \left\langle\overline{u_j^{'}u_k^{'}}\right\rangle\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial x_k} - \left\langle\overline{\theta^{'}u_k^{'}}\right\rangle\frac{\partial\left\langle\overline{u_j}\right\rangle}{\partial x_k} - \frac{\partial\left\langle\overline{u_j^{'}u_k^{'}\theta^{'}}\right\rangle}{\partial x_k} + \frac{g}{T}\left\langle\overline{\theta'^2}\right\rangle\delta_{j3} + \left\langle\overline{p^{'}\frac{\partial\theta^{'}}{\partial x_j}}\right\rangle - (\kappa_T+\nu)\left\langle\overline{\frac{\partial u_j^{'}}{\partial x_k}\frac{\partial\theta^{'}}{\partial x_k}}\right\rangle . \quad (5)$$

Finally, the turbulent temperature variance equation is

$$\frac{\partial\left\langle\overline{\theta'^2}\right\rangle}{\partial t} = 0 = -\left\langle\overline{u_k}\right\rangle\frac{\partial\left\langle\overline{\theta'^2}\right\rangle}{\partial x_k} - 2\left\langle\overline{u_k^{'}\theta^{'}}\right\rangle\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial x_k} - \frac{\partial\left\langle\overline{u_k^{'}\theta'^2}\right\rangle}{\partial x_k} - 2\kappa_T\left\langle\overline{\frac{\partial\theta^{'}}{\partial x_k}\frac{\partial\theta^{'}}{\partial x_k}}\right\rangle \quad . \quad (6)$$

2.2 Modeling Assumptions

Wilson and Shaw *(13)* give the following closure approximation for the pressure drag force in equation 2:

$$\overline{\frac{\partial\ p^{"}}{\partial\ x_i}} \;=\; C_d A\left|\overline{U}\right|\left\langle\overline{u}_i\right\rangle \quad . \quad (7)$$

Here, C_d is the forest canopy drag coefficient (= *0.10*) and A (in units m^2m^{-3}) is the leaf area density. It is further assumed that pressure forces are the main contributor to the total drag from the forest canopy (i.e., viscous drag forces are neglected).

In equation 3, Mellor (*14*) and Mellor and Yamada (*15, 16*) parameterize the triple-velocity products as

$$\left\langle\overline{u_i^{'}u_j^{'}u_k^{'}}\right\rangle = -q\lambda_1\left[\frac{\partial\left\langle\overline{u_j u_k}\right\rangle}{\partial x_i} + \frac{\partial\left\langle\overline{u_i u_j}\right\rangle}{\partial x_k} + \frac{\partial\left\langle\overline{u_i u_k}\right\rangle}{\partial x_j}\right] \quad , \quad (8)$$

where the turbulent kinetic energy (t.k.e.) is $q = \left\langle\overline{u_i^{'}u_i^{'}}\right\rangle^{\frac{1}{2}}$ and λ_1 is a function of mixing length.*

The pressure-velocity gradient terms (i.e., the pressure redistribution terms) in equation 3 can be rewritten as

$$\left\langle\overline{u_i^{'}\frac{\partial p^{'}}{\partial x_k}}\right\rangle + \left\langle\overline{u_k^{'}\frac{\partial p^{'}}{\partial x_i}}\right\rangle = -\left\langle\overline{p^{'}\frac{\partial u_i^{'}}{\partial x_k}}\right\rangle - \left\langle\overline{p^{'}\frac{\partial u_k^{'}}{\partial x_i}}\right\rangle + \left\langle\overline{\frac{\partial p^{'}u_i^{'}}{\partial x_k}}\right\rangle + \left\langle\overline{\frac{\partial p^{'}u_k^{'}}{\partial x_i}}\right\rangle \quad , \quad (9)$$

where

*In these equations, λ_1 through λ_7 are length scales, which contain a set of seven closure constants, i.e., $\lambda_k = a_k l$, where k is an arbitrary index 1–7 and l is the mixing length. Values for these closure constants are as follows: $a_1 = 0.39$, $a_2 = 0.85$, $a_3 = 16.57$, $a_4 = a_6 = 0.23$, $a_5 = 0.74$, and $a_7 = 10.10$ *(11, 13–16)*.

$$\left\langle \overline{p' \frac{\partial u_i'}{\partial x_k}} \right\rangle + \left\langle \overline{p' \frac{\partial u_k'}{\partial x_i}} \right\rangle = -\frac{q}{3\lambda_2}\left[\left\langle \overline{u_i' u_k'} \right\rangle - \delta_{ik} \frac{q^2}{3} \right] + Cq^2 \left[\frac{\partial \left\langle \overline{u_i} \right\rangle}{\partial x_k} + \frac{\partial \left\langle \overline{u_k} \right\rangle}{\partial x_i} \right] \quad , \tag{10}$$

and

$$\left\langle \overline{\frac{\partial p' u_i'}{\partial x_k}} \right\rangle + \left\langle \overline{\frac{\partial p' u_k'}{\partial x_i}} \right\rangle = 0 \quad . \tag{11}$$

This is modeled according to the return-to-isotropy principle, as described by Mellor (*14*) and Donaldson (*17*). In equation 10, C is a constant whose value is about 0.077 (*11*).

Viscous dissipation is assumed isotropic and a function of local t.k.e. intensity. Viscous dissipation, as described by Katul and Albertson (*11*), is parameterized as

$$2\nu \left(\left\langle \overline{\frac{\partial u_i' \partial u_k'}{\partial x_j \partial x_j}} \right\rangle \right) = \frac{2}{3} \frac{q^3}{\lambda_3} \delta_{ik} \quad . \tag{12}$$

In equation 4, the heat source term (S_θ), as described by Meyers and Paw U (*12*), is modeled as

$$S_\theta = 2A \left(\overline{\theta_l} - \overline{\theta} \right) / r_h \quad . \tag{13}$$

Here, A (in units $m^2 m^{-3}$) is the leaf area density, $\left(\overline{\theta_l} - \overline{\theta} \right)$ is the mean leaf surface-to-ambient-air temperature difference, and r_h is the aerodynamic resistance to heat transfer. A 1-D radiative transfer and energy budget algorithm is incorporated into the 2-D model calculation to make it possible to determine the heat source for any time of day. To do this, the formulations outlined by Rachele and Tunick (*18*) are used to calculate the incoming total radiation at the canopy top as a function of latitude, longitude, day of year, and time of day (i.e., these input are needed to determine the solar declination, hour, and zenith angles). Then, the equations provided by Weiss and Norman (*19*) are used to calculate the spectral components for short-wave (i.e., direct beam and diffuse, visible and near-infrared) radiation as a function of the total downward short-wave flux at canopy top because extinction and reflection through the forest canopy are different for each. The remainder of the 1-D radiative transfer subroutine for the forest canopy (i.e., transmission, reflection, absorption, and emission of the solar flux) is derived from the formulations given in the texts by Campbell (*20*) and Campbell and Norman (*21*).

Returning now to equations 5 and 6, the triple-velocity-temperature products contained therein can be expressed in the form described by Mellor (*14*) and Mellor and Yamada (*15, 16*) as

$$\left\langle \overline{u_j' u_k' \theta'} \right\rangle = -q\lambda_4 \left[\frac{\partial \left\langle \overline{u_k' \theta'} \right\rangle}{\partial x_j} + \frac{\partial \left\langle \overline{u_j' \theta'} \right\rangle}{\partial x_k} \right] \tag{14}$$

and

$$\left\langle \overline{u_k' \theta'^2} \right\rangle = -q\lambda_6 \frac{\partial \left\langle \overline{\theta'^2} \right\rangle}{\partial x_k} \quad . \tag{15}$$

In addition, the pressure interaction term can be modeled as

$$\left\langle \overline{p' \frac{\partial \theta'}{\partial x_j}} \right\rangle = -\frac{q}{3\lambda_5} \left\langle \overline{u_j' \theta'} \right\rangle - \frac{1}{3} \frac{g}{T} \left\langle \overline{\theta'^2} \right\rangle \delta_{j3} \quad . \tag{16}$$

Finally, the molecular dissipation of heat can be modeled as

$$2\kappa_T \left\langle \overline{\frac{\partial \theta'}{\partial x_k} \frac{\partial \theta'}{\partial x_k}} \right\rangle = \frac{2q}{\lambda_7} \left\langle \overline{\theta'^2} \right\rangle \quad . \tag{17}$$

This completes the basic equation set and modeling assumptions for the 2-D forest canopy model.

2.3 Second-Order Turbulence Closure Model for 2-D Forest Canopies

The 2-D computer model currently being developed is relatively unique. This is because higher-order closure models reported in the literature (e.g., *11–13, 22*) have generally focused on estimates of 1-D, adiabatic wind flow, and turbulence within and above homogeneous (uniform) forest canopies. In contrast, the new model may be applied night or day to both uniform and nonuniform stands (and forest edges, possibly). The new model, which is based on the earlier works of Katul and Albertson *(11)*, Meyers and Paw U *(12)*, Wilson and Shaw *(13)*, and Donaldson *(17)*, calculates the 2-D, steady state, canopy wind flow, temperatures, turbulent variances, Reynolds stress, and heat flux. The parameterized 2-D model equations for continuity, the mean flow–longitudinal $\langle \overline{u} \rangle$, the mean flow–vertical $\langle \overline{w} \rangle$, Reynolds stress $\langle u'w' \rangle$, longitudinal $\left\langle \overline{u'^2} \right\rangle$, lateral $\left\langle \overline{v'^2} \right\rangle$, and vertical velocity $\left\langle \overline{w'^2} \right\rangle$ variances, the mean temperature $\langle \overline{\theta} \rangle$, vertical heat flux $\left\langle \overline{w'\theta'} \right\rangle$, horizontal heat flux $\left\langle \overline{u'\theta'} \right\rangle$, and the turbulent temperature variance $\left\langle \overline{\theta'^2} \right\rangle$, respectively, are as follows:

$$-\frac{1}{\langle \overline{\rho} \rangle} \frac{\partial \langle \overline{\rho} \rangle}{\partial t} = 0 = \frac{\partial \langle \overline{u} \rangle}{\partial x} + \frac{\partial \langle \overline{w} \rangle}{\partial z} \quad , \tag{18}$$

$$\frac{\partial \langle \overline{u} \rangle}{\partial t} = 0 = -\left\langle \underline{u} \frac{\partial \langle \overline{u} \rangle}{\partial x} \right\rangle - \left\langle \underline{w} \frac{\partial \langle \overline{u} \rangle}{\partial z} \right\rangle - \frac{\partial \left\langle \overline{u'^2} \right\rangle}{\partial x} - \frac{\partial \left\langle \overline{u'w'} \right\rangle}{\partial z} - \left\langle \frac{\partial \overline{p}}{\partial x} \right\rangle - C_d A \left| \overline{U} \right| \langle \overline{u} \rangle \quad , \tag{19}$$

$$\frac{\partial \langle \overline{w} \rangle}{\partial t} = 0 = -\left\langle \underline{u} \frac{\partial \langle \overline{w} \rangle}{\partial x} \right\rangle - \left\langle \underline{w} \frac{\partial \langle \overline{w} \rangle}{\partial z} \right\rangle - \frac{\partial \left\langle \overline{u'w'} \right\rangle}{\partial x} - \frac{\partial \left\langle \overline{w'^2} \right\rangle}{\partial z} - \left\langle \frac{\partial \overline{p}}{\partial z} \right\rangle - C_d A \left| \overline{U} \right| \langle \overline{w} \rangle + \frac{g\theta_0}{T} \quad , \tag{20}$$

$$\frac{\partial\left\langle\overline{u'w'}\right\rangle}{\partial t}=0=-\left\langle\overline{u}\right\rangle\frac{\partial\left\langle\overline{u'w'}\right\rangle}{\partial x}-\left\langle\overline{w}\right\rangle\frac{\partial\left\langle\overline{u'w'}\right\rangle}{\partial z}-\left\langle\overline{u'w'}\right\rangle\frac{\partial\left\langle\overline{u}\right\rangle}{\partial x}-\left\langle\overline{w'^2}\right\rangle\frac{\partial\left\langle\overline{u}\right\rangle}{\partial z}-\left\langle\overline{u'^2}\right\rangle\frac{\partial\left\langle\overline{w}\right\rangle}{\partial x}-\left\langle\overline{u'w'}\right\rangle\frac{\partial\left\langle\overline{w}\right\rangle}{\partial z}$$
$$+\frac{\partial}{\partial x}\left(q\lambda_1\left(2\frac{\partial\left\langle\overline{u'w'}\right\rangle}{\partial x}+\frac{\partial\left\langle\overline{u'^2}\right\rangle}{\partial z}\right)\right)+\frac{\partial}{\partial z}\left(q\lambda_1\left(\frac{\partial\left\langle\overline{w'^2}\right\rangle}{\partial x}+2\frac{\partial\left\langle\overline{u'w'}\right\rangle}{\partial z}\right)\right)+\frac{g}{T}\left\langle\overline{u'\theta'}\right\rangle-\frac{q}{3\lambda_2}\left\langle\overline{u'w'}\right\rangle \tag{21}$$
$$+Cq^2\left(\frac{\partial\left\langle\overline{u}\right\rangle}{\partial z}+\frac{\partial\left\langle\overline{w}\right\rangle}{\partial x}\right)+\left\langle\overline{w}\right\rangle C_d A\left|\overline{U}\right|\left\langle\overline{u}\right\rangle+\left\langle\overline{u}\right\rangle C_d A\left|\overline{U}\right|\left\langle\overline{w}\right\rangle\ ,$$

$$\frac{\partial\left\langle\overline{u'^2}\right\rangle}{\partial t}=0=-\left\langle\overline{u}\right\rangle\frac{\partial\left\langle\overline{u'^2}\right\rangle}{\partial x}-\left\langle\overline{w}\right\rangle\frac{\partial\left\langle\overline{u'^2}\right\rangle}{\partial z}-2\left\langle\overline{u'^2}\right\rangle\frac{\partial\left\langle\overline{u}\right\rangle}{\partial x}-2\left\langle\overline{u'w'}\right\rangle\frac{\partial\left\langle\overline{u}\right\rangle}{\partial z}+\frac{\partial}{\partial x}\left(3q\lambda_1\frac{\partial\left\langle\overline{u'^2}\right\rangle}{\partial x}\right)$$
$$+\frac{\partial}{\partial z}\left(q\lambda_1\left(2\frac{\partial\left\langle\overline{u'w'}\right\rangle}{\partial x}+\frac{\partial\left\langle\overline{u'^2}\right\rangle}{\partial z}\right)\right)-\frac{q}{3\lambda_2}\left(\left\langle\overline{u'^2}\right\rangle-\frac{q^2}{3}\right)+2Cq^2\frac{\partial\left\langle\overline{u}\right\rangle}{\partial x}+2C_d A\left|\overline{U}\right|\left\langle\overline{u}\right\rangle^2-\frac{2}{3}\frac{q^3}{\lambda_3}\ , \tag{22}$$

$$\frac{\partial\left\langle\overline{v'^2}\right\rangle}{\partial t}=0=-\left\langle\overline{u}\right\rangle\frac{\partial\left\langle\overline{v'^2}\right\rangle}{\partial x}-\left\langle\overline{w}\right\rangle\frac{\partial\left\langle\overline{v'^2}\right\rangle}{\partial z}+\frac{\partial}{\partial x}\left(q\lambda_1\frac{\partial\left\langle\overline{v'^2}\right\rangle}{\partial x}\right)+\frac{\partial}{\partial z}\left(q\lambda_1\frac{\partial\left\langle\overline{v'^2}\right\rangle}{\partial z}\right)-\frac{q}{3\lambda_2}\left(\left\langle\overline{v'^2}\right\rangle-\frac{q^2}{3}\right)-\frac{2}{3}\frac{q^3}{\lambda_3}\quad , \tag{23}$$

$$\frac{\partial\left\langle\overline{w'^2}\right\rangle}{\partial t}=0=-\left\langle\overline{u}\right\rangle\frac{\partial\left\langle\overline{w'^2}\right\rangle}{\partial x}-\left\langle\overline{w}\right\rangle\frac{\partial\left\langle\overline{w'^2}\right\rangle}{\partial z}-2\left\langle\overline{u'w'}\right\rangle\frac{\partial\left\langle\overline{w}\right\rangle}{\partial x}-2\left\langle\overline{w'^2}\right\rangle\frac{\partial\left\langle\overline{w}\right\rangle}{\partial z}+\frac{\partial}{\partial x}\left(q\lambda_1\left(2\frac{\partial\left\langle\overline{u'w'}\right\rangle}{\partial z}+\frac{\partial\left\langle\overline{w'^2}\right\rangle}{\partial x}\right)\right)$$
$$+\frac{\partial}{\partial z}\left(3q\lambda_1\frac{\partial\left\langle\overline{w'^2}\right\rangle}{\partial z}\right)+2\frac{g}{T}\left\langle\overline{w'\theta'}\right\rangle-\frac{q}{3\lambda_2}\left(\left\langle\overline{w'^2}\right\rangle-\frac{q^2}{3}\right)+2Cq^2\frac{\partial\left\langle\overline{w}\right\rangle}{\partial z}+2C_d A\left|\overline{U}\right|\left\langle\overline{w}\right\rangle^2-\frac{2}{3}\frac{q^3}{\lambda_3}\ , \tag{24}$$

$$\frac{d\left\langle\overline{\theta}\right\rangle}{dt}=0=-\left\langle\overline{u}\right\rangle\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial x}-\left\langle\overline{w}\right\rangle\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial z}-\frac{\partial\left\langle\overline{u'\theta'}\right\rangle}{\partial x}-\frac{\partial\left\langle\overline{w'\theta'}\right\rangle}{\partial z}+S_\theta\quad , \tag{25}$$

$$\frac{\partial\left\langle\overline{w'\theta'}\right\rangle}{\partial t}=0=-\left\langle\overline{u}\right\rangle\frac{\partial\left\langle\overline{w'\theta'}\right\rangle}{\partial x}-\left\langle\overline{w}\right\rangle\frac{\partial\left\langle\overline{w'\theta'}\right\rangle}{\partial z}-\left\langle\overline{u'w'}\right\rangle\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial x}-\left\langle\overline{w'^2}\right\rangle\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial z}-\left\langle\overline{u'\theta'}\right\rangle\frac{\partial\left\langle\overline{w}\right\rangle}{\partial x}-\left\langle\overline{w'\theta'}\right\rangle\frac{\partial\left\langle\overline{w}\right\rangle}{\partial z}$$
$$+\frac{\partial}{\partial x}\left(q\lambda_4\left(\frac{\partial\left\langle\overline{u'\theta'}\right\rangle}{\partial z}+\frac{\partial\left\langle\overline{w'\theta'}\right\rangle}{\partial x}\right)\right)+\frac{\partial}{\partial z}\left(2q\lambda_4\frac{\partial\left\langle\overline{w'\theta'}\right\rangle}{\partial z}\right)+\frac{2}{3}\frac{g}{T}\left\langle\overline{\theta'^2}\right\rangle-\frac{q}{3\lambda_5}\left\langle\overline{w'\theta'}\right\rangle\ , \tag{26}$$

$$\frac{\partial\left\langle\overline{u'\theta'}\right\rangle}{\partial t}=0=-\left\langle\overline{u}\right\rangle\frac{\partial\left\langle\overline{u'\theta'}\right\rangle}{\partial x}-\left\langle\overline{w}\right\rangle\frac{\partial\left\langle\overline{u'\theta'}\right\rangle}{\partial z}-\left\langle\overline{u'^2}\right\rangle\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial x}-\left\langle\overline{u'w'}\right\rangle\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial z}-\left\langle\overline{u'\theta'}\right\rangle\frac{\partial\left\langle\overline{u}\right\rangle}{\partial x}-\left\langle\overline{w'\theta'}\right\rangle\frac{\partial\left\langle\overline{u}\right\rangle}{\partial z}$$
$$+\frac{\partial}{\partial x}\left(2q\lambda_4\frac{\partial\left\langle\overline{u'\theta'}\right\rangle}{\partial x}\right)+\frac{\partial}{\partial z}\left(q\lambda_4\left(\frac{\partial\left\langle\overline{w'\theta'}\right\rangle}{\partial x}+\frac{\partial\left\langle\overline{u'\theta'}\right\rangle}{\partial z}\right)\right)-\frac{q}{3\lambda_5}\left\langle\overline{u'\theta'}\right\rangle\ , \tag{27}$$

and

$$\frac{\partial\left\langle\overline{\theta'^2}\right\rangle}{\partial t}=0=-\left\langle\overline{u}\right\rangle\frac{\partial\left\langle\overline{\theta'^2}\right\rangle}{\partial x}-\left\langle\overline{w}\right\rangle\frac{\partial\left\langle\overline{\theta'^2}\right\rangle}{\partial z}-2\left\langle\overline{u'\theta'}\right\rangle\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial x}-2\left\langle\overline{w'\theta'}\right\rangle\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial z}$$
$$+\frac{\partial}{\partial x}\left(q\lambda_6\frac{\partial\left\langle\overline{\theta'^2}\right\rangle}{\partial x}\right)+\frac{\partial}{\partial z}\left(q\lambda_6\frac{\partial\left\langle\overline{\theta'^2}\right\rangle}{\partial z}\right)-\frac{2q}{\lambda_7}\left\langle\overline{\theta'^2}\right\rangle\ . \tag{28}$$

In the model, mean pressure is assumed approximately hydrostatic. In contrast, the kinematic (fluctuation) pressure $\left\langle\overline{p}\right\rangle$ is determined, as discussed in the text by Ferziger and Perić (*23*), by taking the divergence of the mean flow equations, i.e.,

$$\frac{\partial}{\partial x}\left(\frac{\partial\left\langle\overline{u}\right\rangle}{\partial t}\right)=0=-\frac{\partial}{\partial x}\left\langle\overline{u}\frac{\partial\left\langle\overline{u}\right\rangle}{\partial x}\right\rangle-\frac{\partial}{\partial x}\left\langle\overline{w}\frac{\partial\left\langle\overline{u}\right\rangle}{\partial z}\right\rangle-\frac{\partial^2\left\langle\overline{u'^2}\right\rangle}{\partial x^2}-\frac{\partial}{\partial x}\left(\frac{\partial\left\langle\overline{u'w'}\right\rangle}{\partial z}\right)$$
$$-\left\langle\frac{\partial^2\overline{p}}{\partial x^2}\right\rangle-C_dA|U|\frac{\partial\left\langle\overline{u}\right\rangle}{\partial x}\ , \tag{29}$$

and

$$\frac{\partial}{\partial z}\left(\frac{\partial\left\langle\overline{w}\right\rangle}{\partial t}\right)=0=-\frac{\partial}{\partial z}\left\langle\overline{u}\frac{\partial\left\langle\overline{w}\right\rangle}{\partial x}\right\rangle-\frac{\partial}{\partial z}\left\langle\overline{w}\frac{\partial\left\langle\overline{w}\right\rangle}{\partial z}\right\rangle-\frac{\partial}{\partial z}\left(\frac{\partial\left\langle\overline{u'w'}\right\rangle}{\partial x}\right)-\frac{\partial^2\left\langle\overline{w'^2}\right\rangle}{\partial z^2}$$
$$-\left\langle\frac{\partial^2\overline{p}}{\partial z^2}\right\rangle+\frac{g}{T}\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial z}-C_dA|U|\frac{\partial\left\langle\overline{w}\right\rangle}{\partial z}\ , \tag{30}$$

which, after some rearranging and cancellation of terms due to continuity, yields Poisson's equation (i.e., pressure-velocity coupling) as

$$\left\langle\frac{\partial^2\overline{p}}{\partial z^2}\right\rangle+\left\langle\frac{\partial^2\overline{p}}{\partial x^2}\right\rangle=-\frac{\partial^2\left\langle\overline{u'^2}\right\rangle}{\partial x^2}-\frac{\partial}{\partial x}\left(\frac{\partial\left\langle\overline{u'w'}\right\rangle}{\partial z}\right)-\frac{\partial}{\partial z}\left(\frac{\partial\left\langle\overline{u'w'}\right\rangle}{\partial x}\right)-\frac{\partial^2\left\langle\overline{w'^2}\right\rangle}{\partial z^2}+\frac{g}{T}\frac{\partial\left\langle\overline{\theta}\right\rangle}{\partial z}\quad . \tag{31}$$

Also, note that the lateral velocity variance $\left\langle\overline{v'^2}\right\rangle$ is calculated in the current 2-D model (equation 23) to support the turbulence (t.k.e.) closure approximations for the triple product and dissipation terms.

2.4 Numerical Methods

A computational grid consisting of 10 × 60 points is chosen, where the horizontal grid spacing (Δx) is 50 m and the vertical grid spacing (Δz) is 0.5 m. Hence, the model domain is 500 × 30 m. Vertical derivatives are solved using a lower-order central differencing scheme (*23*). The first derivatives can be expressed as

$$\frac{\partial \phi_{(i,j)}}{\partial z} = \frac{\phi_{(i,j+1)} - \phi_{(i,j-1)}}{2\Delta z} \quad , \tag{32}$$

and the second derivatives as

$$\frac{\partial^2 \phi_{(i,j)}}{\partial z^2} = \frac{\phi_{(i,j-1)} - 2\phi_{(i,j)} + \phi_{(i,j+1)}}{(\Delta z)^2} \quad . \tag{33}$$

Here, *i* is the horizontal grid index and *j* is the vertical grid index. In contrast, the horizontal derivatives are solved using a lower-order upwind differencing scheme (*23*), such that the first derivatives are computed as

$$\frac{\partial \phi_{(i,j)}}{\partial x} = \frac{\phi_{(i,j)} - \phi_{(i-1,j)}}{\Delta x} \quad , \tag{34}$$

and the second derivatives are given as

$$\frac{\partial^2 \phi_{(i,j)}}{\partial x^2} = \frac{\phi_{(i,j)} - 2\phi_{(i-1,j)} + \phi_{(i-2,j)}}{(\Delta x)^2} \quad . \tag{35}$$

A set of simultaneous equations for each of 12 computed variables is then solved iteratively using the Thomas algorithm, a tridiagonal matrix solver (*24*). In addition, the solution implements a relaxation scheme for all the computed variables, similar to that described by Wilson (*22*), i.e.,

$$\phi = a\phi^n + (1-a)\phi^{n-1} \quad , \tag{36}$$

where $a = 0.05$, n is the current iteration, and $n - 1$ is the previous iteration. The relaxation scheme primarily affects the rate at which the solution converges, not how well the solution converges.

Then, based on the earlier works of Meyers and Paw U (*12*) and Katul and Albertson (*11*), the following (model) top and bottom boundary conditions are applied:

At $z = 0$:

$\langle \overline{u} \rangle = \langle \overline{w} \rangle = 0$; $\langle \overline{\theta} \rangle = 303.15$;

$\langle \overline{p} \rangle = 0$;

$\langle \overline{u'w'} \rangle = 0$; $\langle \overline{w'\theta'} \rangle = \langle \overline{u'\theta'} \rangle = 0$;

$\left\langle \overline{u'^2} \right\rangle = \left\langle \overline{v'^2} \right\rangle = \left\langle \overline{w'^2} \right\rangle = \left\langle \overline{\theta'^2} \right\rangle = 0$;

$u_* = \left| \left\langle \overline{u'w'} \right\rangle_{(i,2)} \right|^{0.5}$ (friction velocity at $z = 2\Delta z$);

$\theta_* = \dfrac{\left\langle \overline{w'\theta'} \right\rangle_{(i,2)}}{u_*}$ (potential temperature scaling constant at $z = 2\Delta z$);

$\dfrac{\partial \left\langle \overline{u} \right\rangle}{\partial z} = \dfrac{u_*}{k 2\Delta z}$ and $\dfrac{\partial \left\langle \overline{\theta} \right\rangle}{\partial z} = \dfrac{\theta_*}{k 2\Delta z}$ ($k = 0.4$ is von Karman's constant);

$\dfrac{\partial \left\langle \overline{w} \right\rangle}{\partial z} = 0$ and $\dfrac{\partial \left\langle \overline{p} \right\rangle}{\partial z} = 0$.

At $z = 30$ *m*:

$\dfrac{\partial \left\langle \overline{u} \right\rangle}{\partial z} = \dfrac{u_*}{k(z-d)}$ and $\dfrac{\partial \left\langle \overline{\theta} \right\rangle}{\partial z} = \dfrac{\theta_*}{k(z-d)}$

(d is displacement height ($d \approx \frac{2}{3} h$) and h is canopy height);

$u_* = \dfrac{k\left(U^{top} - \left\langle \overline{u} \right\rangle_{(i,h+10\Delta z)} \right)}{\log\left(\dfrac{z-d}{h+10\Delta z - d} \right)}$ ($U^{top} = 7.0 \ \ ms^{-1}$);

$Q_h = \int_0^h S_\theta \ dz$ (kinematic heat flux, $Q_h = \left\langle \overline{w'\theta'} \right\rangle$);

$\theta_* = \dfrac{-Q_h}{u_*}$ (potential temperature scaling constant);

$\dfrac{\partial \left\langle \overline{w} \right\rangle}{\partial z} = 0$ and $\dfrac{\partial \left\langle \overline{p} \right\rangle}{\partial z} = 0$;

$\left\langle \overline{u'w'} \right\rangle = -u_*^2$; $\left\langle \overline{w'\theta'} \right\rangle = Q_h$; and $\left\langle \overline{u'\theta'} \right\rangle = -3.0 \ Q_h$;

$\left\langle \overline{u'^2} \right\rangle = 3.5 \ u_*^2$; $\left\langle \overline{v'^2} \right\rangle = 1.5 \ u_*^2$; $\left\langle \overline{w'^2} \right\rangle = 1.5 \ u_*^2$; and $\left\langle \overline{\theta'^2} \right\rangle = 4.0 \ \theta_*^2$.

In addition, zero-gradient conditions are assumed at the lateral boundaries. However, a weighted smoother, similar to that proposed by Mahrer and Pielke (*25*), is applied to the first three and last three horizontal grid points to reduce undesired boundary effects, i.e.,

$$\phi_{(i,j)} = 0.5\phi_{(i,j)} + 0.25\left(\phi_{(i+1,j)} + \phi_{(i-1,j)}\right) \quad . \tag{37}$$

Finally, lower-order Newton-Cotes formulas for numerical integration, i.e., the trapezoidal rule and Simpson's one-third rule, are applied to derive the wind flow streamlines.

2.5 Forest Canopy Architecture

As shown in sections 2.2 and 2.3, canopy architecture plays an important role in defining the momentum and heat flux divergence through the forest layer. Following the discussions by Massman (26) and Meyers et al. (27), it was suggested that forest canopies may conform to one of three general leaf area distribution profiles, as shown in Figure 1. It is clear that leaf area distributions are not always symmetric about the layer of maximum foliage density (like profile-1) but may be more often skewed upward toward the top of the forest canopy. By definition, leaf area index is $LAI = \int_0^h A(z)\, dz$, where *A(z)* is the leaf area density through the small vertical layer between *z* and $z + \Delta z$ per unit surface area of ground below *(28)*. Values for leaf area index for forests vary but have been reported most often in the range LAI = 1 to 5 *(29)*. In section 3, 2-D model results will be shown for the case corresponding to profile-2. Also, in this study LAI = 3.0.

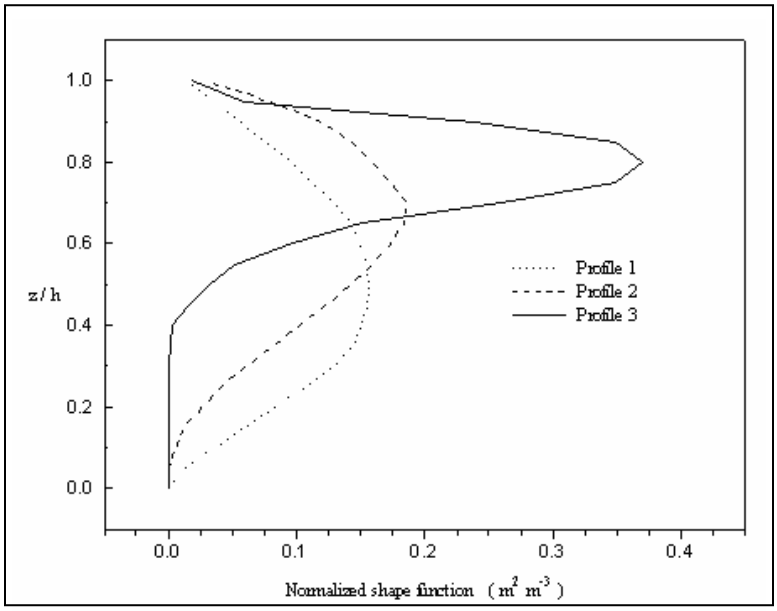


Figure 1. Normalized vertical profiles of leaf area density for forest canopies.

3. Model Results

3.1 Uniform Forest Stands

In this section, several initial model results are presented for a 2-D uniform forest stand. Figure 2 shows the calculated fields for horizontal wind velocity, $\langle \overline{u} \rangle$, in units ms^{-1}, and air temperature, $\langle \overline{\theta} \rangle$, in units °C. For this example, upper level (i.e., model top) wind velocity is

u_{max} = 7.0 ms^{-1}, leaf area index is LAI = 3, and forest canopy height is h = 10 m. In contrast, Figure 3 shows individual profiles for wind velocity and air temperature derived from the current 2-D calculation in comparison to profiles derived from an earlier 1-D modeling study *(1, 2)*. Profile results from the 1-D and 2-D models agree reasonably well. Small differences are due (possibly) to the expanded numerical grid. Finally, Figure 4 shows some spurious oscillations that result when Δx = 20 *m*. Here, it is likely that the solution is unstable because $\Delta x/Z(N) < 1$, where *Z(N)* is the height at the model top. Based on an analysis of the equation set and several additional calculations (not shown), this condition for stability appears to be valid. The computed variable in Figure 4 is horizontal wind velocity, $\langle \overline{u} \rangle$, in units ms^{-1}.

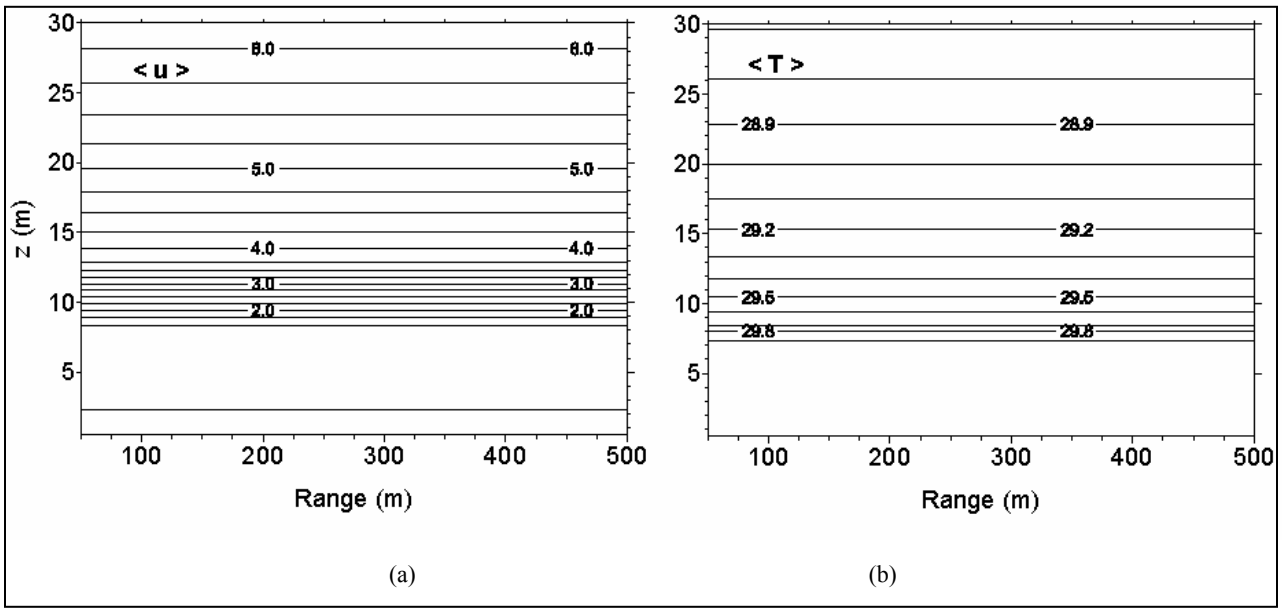


Figure 2. Model results for uniform forest stands: (a) horizontal wind velocity, $\langle \overline{u} \rangle$, in units ms^{-1}, and (b) air temperature, $\langle \overline{\theta} \rangle$, in units °C. For this example, canopy height (h) is 10 m.

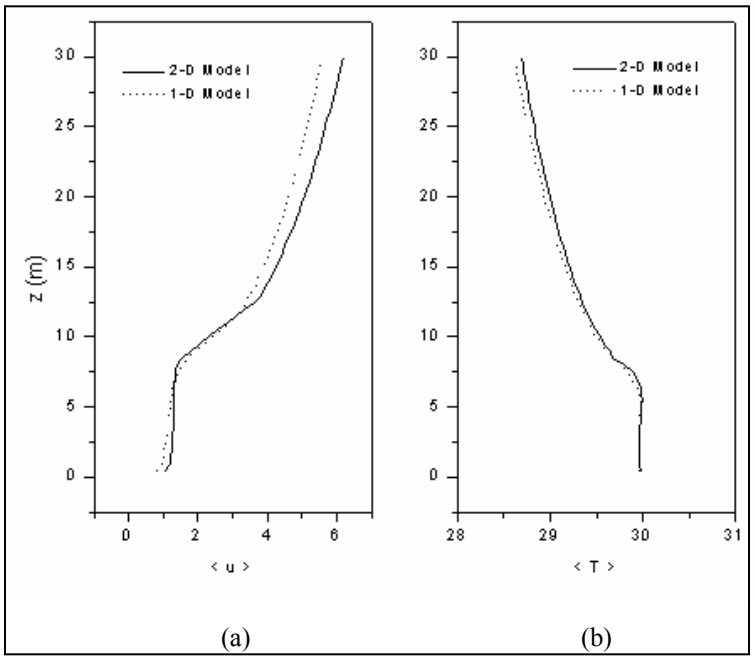


Figure 3. Profiles of horizontal wind velocity ($\langle \bar{u} \rangle$, in units ms^{-1}) and air temperature ($\langle \bar{\theta} \rangle$, in units °C) derived from the current 2-D calculation shown in comparison to 1-D profiles derived from an earlier study.

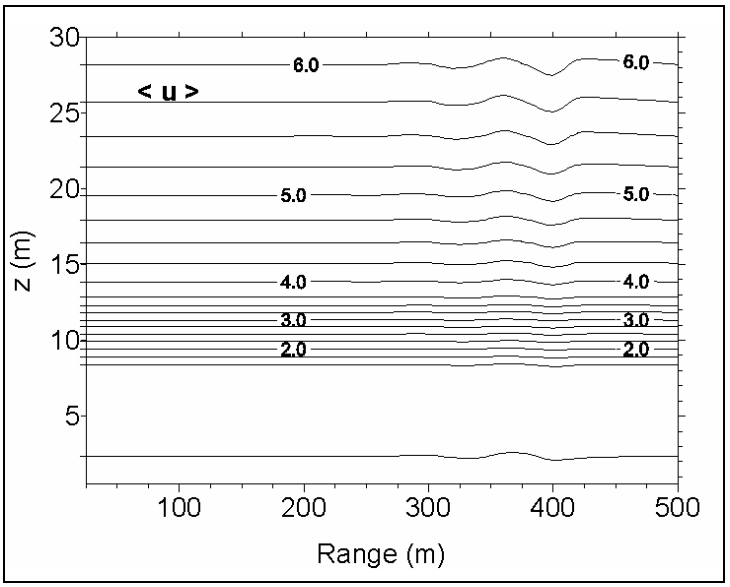


Figure 4. Spurious oscillations due to numerical instability in the 2-D model results for uniform forest stands. The computed variable is horizontal wind velocity, $\langle \bar{u} \rangle$, in units ms^{-1}.

3.2 Nonuniform Forest Stands

Figures 5–7 show the current 2-D model results for a nonuniform forest stand, i.e., which contains a single step change in canopy height. The computed variables in Figure 5 are horizontal wind velocity, $\langle \overline{u} \rangle$, in units ms^{-1}, and the wind flow streamlines. A single step change in canopy height (h) at one grid increment past the midpoint (i.e., $X/2 + \Delta x$) is shown via open rectangles, where $h = 8\ m$ on the left side and $h = 10\ m$ on the right side. The computed variables in Figure 6 are vertical wind velocity, $\langle \overline{w} \rangle$, in units ms^{-1}, and kinematic (fluctuation) pressure $\langle \overline{p} \rangle$, in units $m^{-2}s^{-2}$. The computed variables in Figure 7 are air temperature, $\langle \overline{\theta} \rangle$, in units °C, and the effective speed of sound, $\langle \overline{C_{eff}} \rangle$, in units ms^{-1}. Finally, Figure 8 shows spurious oscillations that result when $\Delta x = 20\ m$ instead of $\Delta x = 50\ m$. Here again, a computational instability is brought about (possibly) because $\Delta x/Z(N) < 1$.

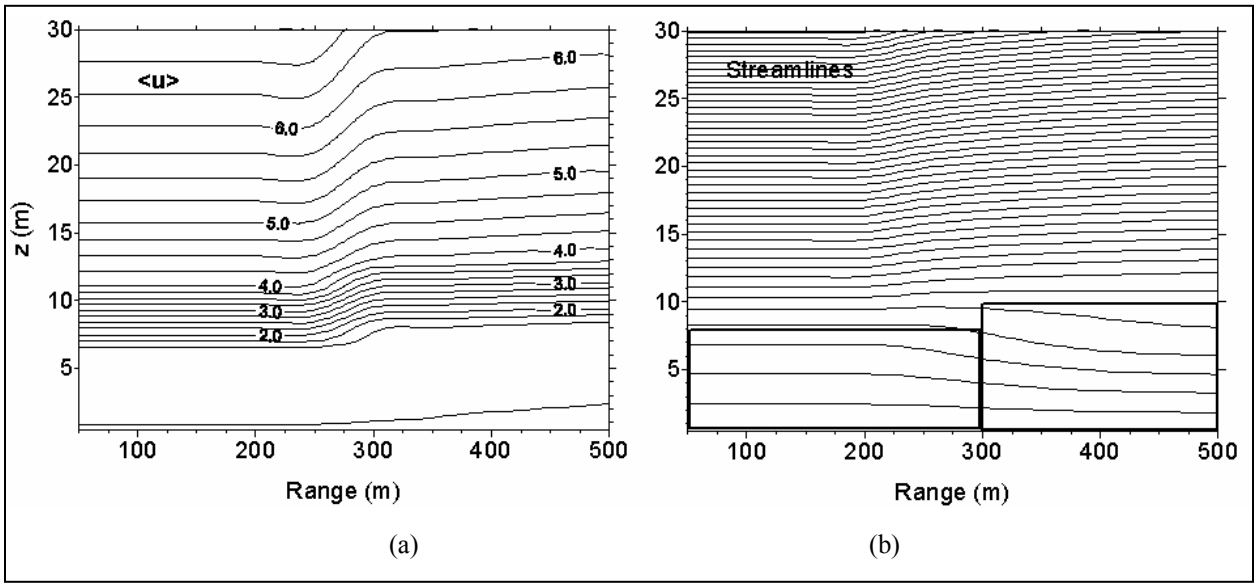
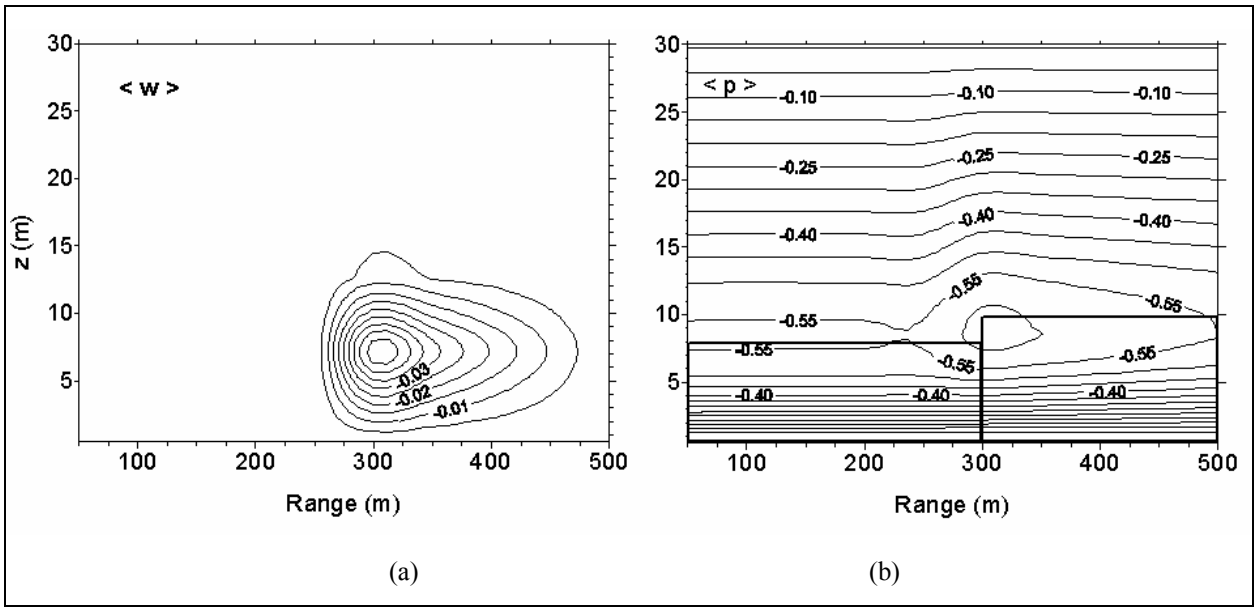
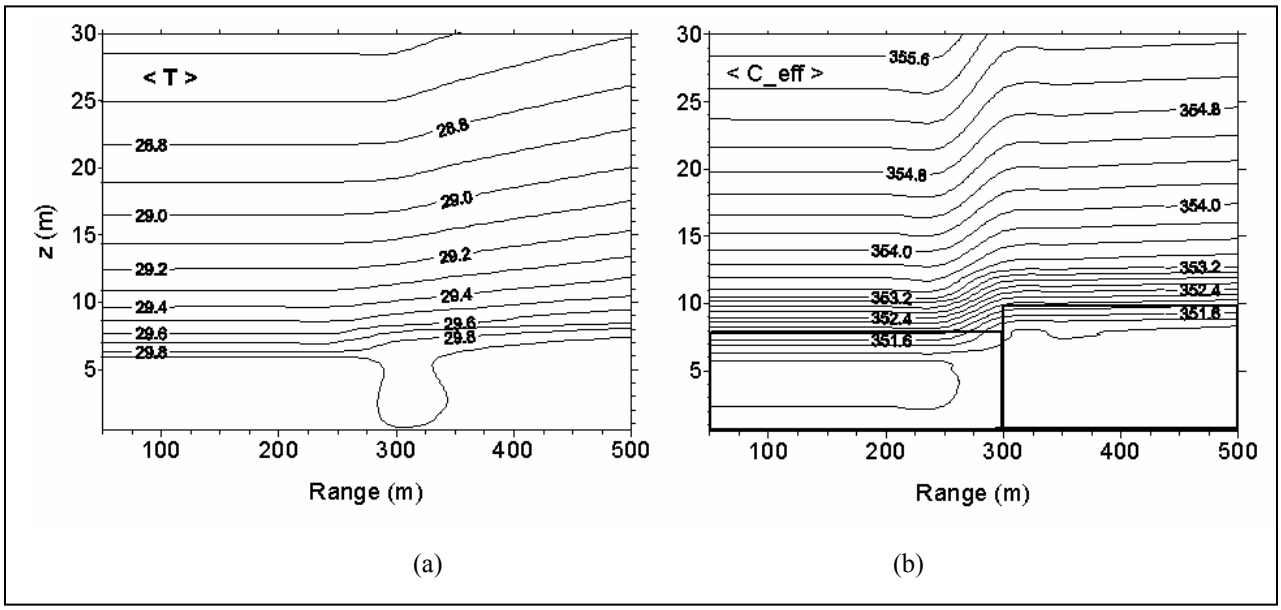


Figure 5. Model results for nonuniform forest stands, i.e., those that contain a single step change in canopy height: (a) horizontal wind velocity, $\langle \overline{u} \rangle$, in units ms^{-1}, and (b) wind flow streamlines. A single step change in canopy height at $X/2 + \Delta x$ is shown using open rectangles, where $h = 8$ m on the left side and $h = 10$ m on the right side.



(a) (b)

Figure 6. Model results for nonuniform forest stands: (a) vertical wind velocity, $\langle \overline{w} \rangle$, in units ms^{-1} and (b) kinematic (fluctuation) pressure $\langle \overline{p} \rangle$, in units $m^{-2}s^{-2}$. A single step change in canopy height occurs at $X/2 + \Delta x$, where h = 8 m on the left side and h = 10 m on the right side.



(a) (b)

Figure 7. Model results for nonuniform forest stands: (a) air temperature, $\langle \overline{\theta} \rangle$, in units °C and (b) effective speed of sound, $\langle \overline{C_{eff}} \rangle$, in units ms^{-1}. A single step change in canopy height occurs at $X/2 + \Delta x$, where h = 8 m on the left side and h = 10 m on the right side.

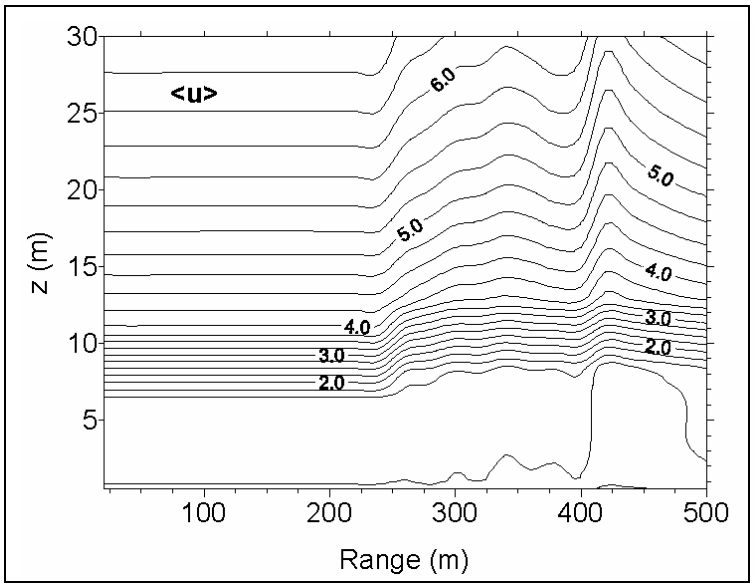


Figure 8. Spurious oscillations due to numerical instability in the 2-D model results for nonuniform forest stands. The computed variable is horizontal wind velocity, $\langle \overline{u} \rangle$, in units ms^{-1}.

Figures 9–11 show the current 2-D model results for a second nonuniform forest stand. In this example, a larger step in canopy height is incorporated at the lower boundary, where $h = 4$ m on the left side and $h = 10$ m on the right side. The computed variables in Figure 9 are horizontal wind velocity, $\langle \overline{u} \rangle$, in units ms^{-1}, and the wind flow streamlines. The computed variables in Figure 10 are vertical wind velocity, $\langle \overline{w} \rangle$, in units ms^{-1}, and kinematic (fluctuation) pressure $\langle \overline{p} \rangle$, in units $m^{-2}s^{-2}$. The computed variables in Figure 11 are air temperature, $\langle \overline{\theta} \rangle$, in units °C, and the effective speed of sound, $\langle \overline{C_{eff}} \rangle$, in units ms^{-1}. Finally, Figure 12 shows that care needs to be taken when applying numerical (integration) schemes across sharp discontinuities (e.g., to calculate the streamlines). Here, the use of Simpson's one-third rule for both integrals, i.e., $\int d\psi = -\int w dx + \int u dz$, resulted in a computational instability (i.e., $2\Delta x$ numerical waves). Upon further analysis, these were found to be contained mainly in the solution to $-\int w dx$. A stable solution was later obtained by applying a simpler trapezoidal rule to solve this integral. Note that while $\Delta x = 50\ \ m$ in Figure 12, the number of horizontal grid points is expanded (i.e., *MPT* = 40).

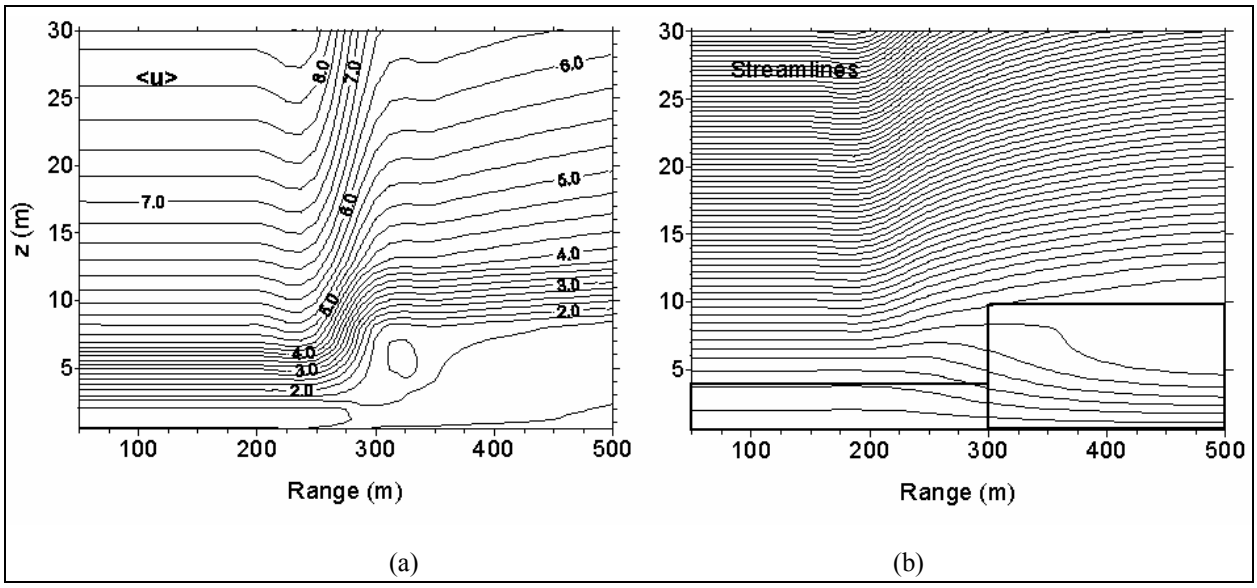


Figure 9. Model results for nonuniform forest stands, i.e., those that contain a single step change in canopy height: (a) horizontal wind velocity, $\langle \overline{u} \rangle$, in units ms^{-1}, and (b) wind flow streamlines. A single step change in canopy height at $X/2 + \Delta x$ is shown using open rectangles, where $h = 4$ m on the left side and $h = 10$ m on the right side.

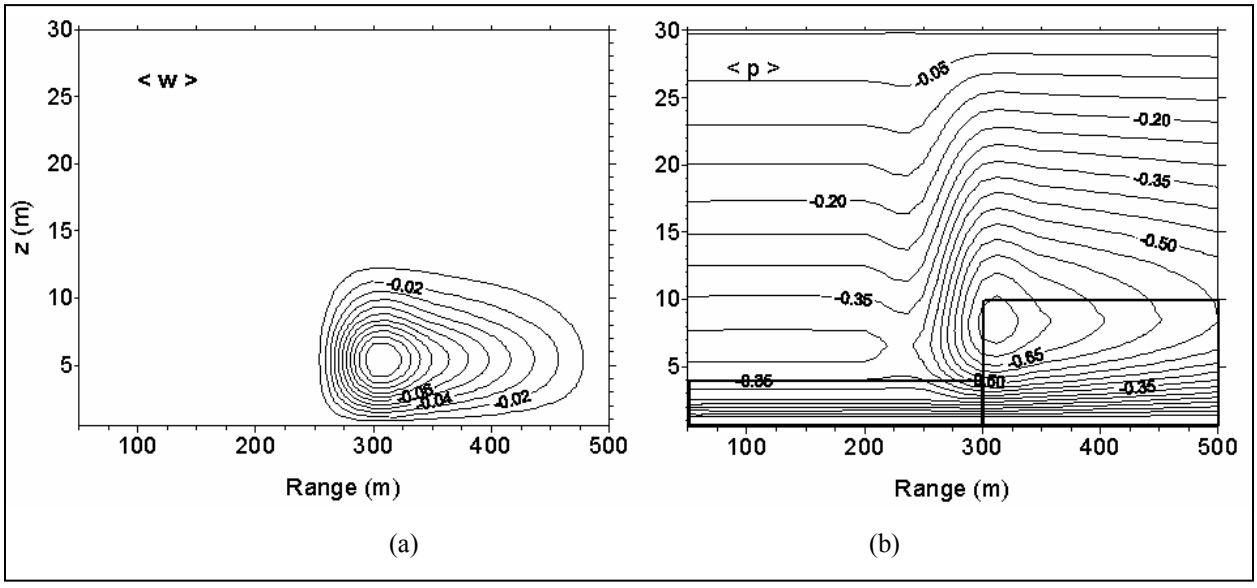


Figure 10. Model results for nonuniform forest stands: (a) vertical wind velocity, $\langle \overline{w} \rangle$, in units ms^{-1}, and (b) kinematic (fluctuation) pressure $\langle \overline{p} \rangle$, in units $\mathrm{m}^{-2}\mathrm{s}^{-2}$. A single step change in canopy height occurs at $X/2 + \Delta x$, where $h = 4$ m on the left side and $h = 10$ m on the right side.

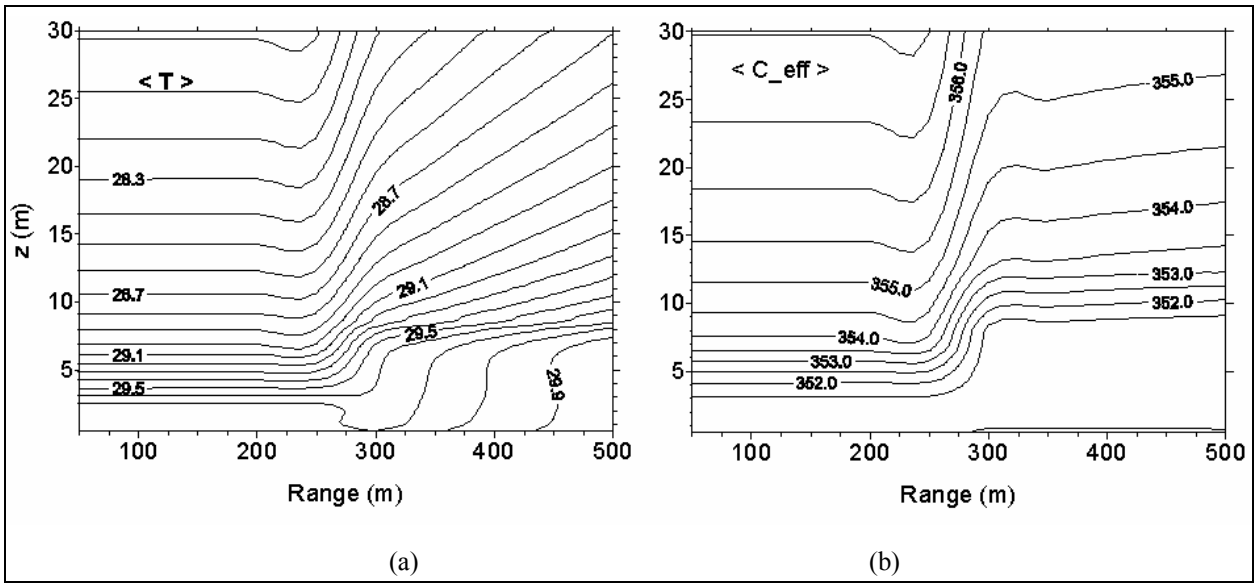


Figure 11. Model results for nonuniform forest stands: (a) air temperature, $\langle \overline{\theta} \rangle$, in units °C, and (b) effective speed of sound, $\langle \overline{C_{eff}} \rangle$, in units ms^{-1}. A single step change in canopy height occurs at $X/2 + \Delta x$, where $h = 4$ m on the left side and $h = 10$ m on the right side.

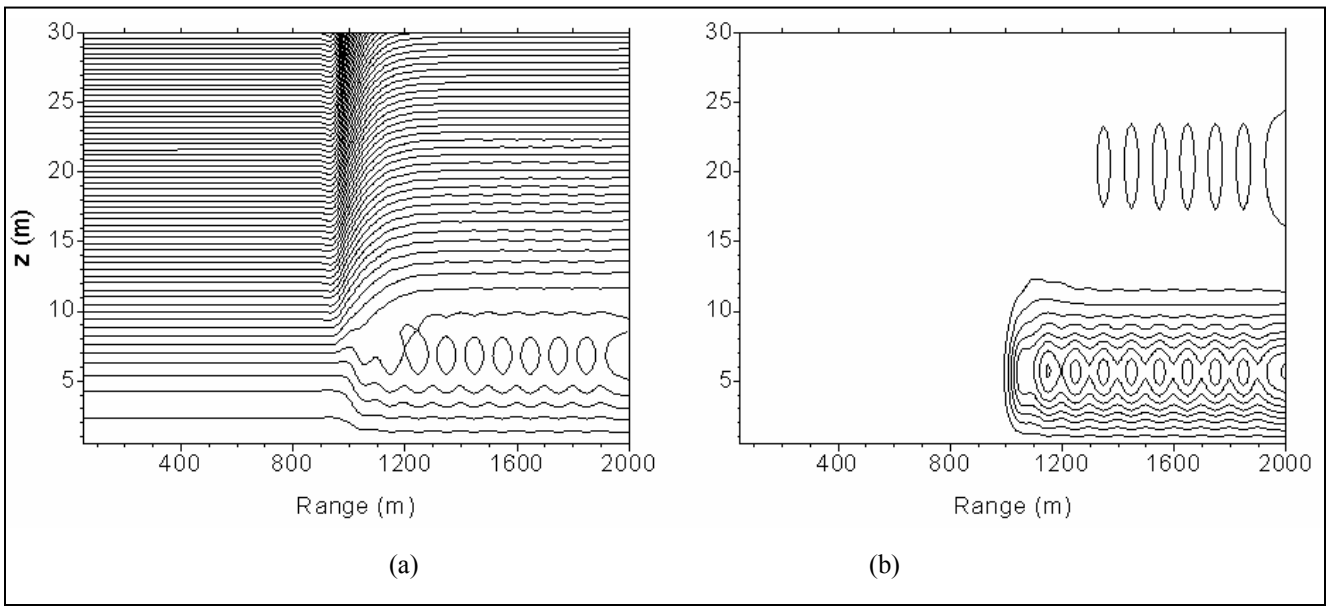


Figure 12. Numerical instability, i.e., $2\Delta x$ waves, in the computation of wind flow streamlines for nonuniform forest stands: (a) $\int d\psi = -\int w dx + \int u dz$ and (b) $\int d\psi = -\int w dx$ only. A single step change in canopy height occurs at $X/2 + \Delta x$, where $h = 4$ m on the left side and $h = 10$ m on the right side.

4. Summary and Conclusions

A new and relatively unique, physics-based, meteorological computer model for 2-D forest canopy wind flow, temperature, and turbulence calculations has been developed. The current 2-D model is based on the same basic conservation-law equations and (second-order turbulence closure) modeling assumptions that were implemented in an earlier 1-D model study to mathematically represent the mechanical and thermodynamic influences on the speed of sound in the forest environment. The 2-D computer model has been implemented in Fortran. Valid numerical techniques from earlier works have been applied to solve for each of twelve computed variables, to include the mean flow vertical velocity and the kinematic (fluctuation) pressure. The horizontal derivatives were solved using lower-order, upwind differencing. The vertical derivatives were solved using lower-order, central differencing. Second-order, ordinary differential equations for the profiles were solved using a tridiagonal matrix algorithm. Thus, several satisfactory solutions were achieved for uniform and nonuniform forest stands (for coarse horizontal grid spacing, i.e., $\Delta x = 50$ m). In addition, a valid condition for numerical stability was determined, as $\Delta x/Z(N) > 1$.

In future modeling works, we will continue to investigate these and alternate numerical schemes, which are accurate, fast, and robust, to solve the computed fields within and above a realistic forest canopies, particularly those containing sharp discontinuities at the lower boundary (e.g., forest edges [*30*]). Also, the presence of hills may significantly alter the flow field inside canopies (*31*). Most turbulence models, including the model described herein, are for flat surfaces. Therefore, additional 2-D forest canopy models may begin to consider uneven terrain.

5. References

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